Structural Performance of Space Trusses under Fire

Amr M. Ibrahim¹, Ahmed A. Elshafey², Boshra A. Eltaly ³ and Kamel S. Kandil³

1 Ph.D. Student, Civil Engineering Department, Faculty of Engineering, Minoufia University, Gamal Abdul-Nasser Street, Minoufia, Egypt, E-mail: amribrahim@mail.com

2 Lecturer, Civil Engineering Department, Faculty of Engineering, Minoufia University, Gamal Abdul-

Nasser Street, Minoufia, Egypt, E-mail: abasahmed68@hotmail.com

3Associate Professor, Civil Engineering Department, Faculty of Engineering, Minoufia University, Gamal

Abdul-Nasser Street, Minoufia, Egypt, E-mail: Boushra_Eltaly@yahoo.com

4Professor, Civil Engineering Department, Faculty of Engineering, Minoufia University, Gamal Abdul-Nasser Street, Minoufia, Egypt, E-mail: kandeel@kandeel.org

ABSTRACT

An experimental program was designed in the current work to capture the structural behavior of two different space trusses under static loads and under increasing the temperature at the truss nodes up to failure. The results of the two space trusses tested include the central deflection of the truss and the axial forces in truss members under the applied loads and temperature. Also analytical studies based on finite element method for the two tested trusses are introduced in this paper. A comparison between experimental work and analytical study is presented and discussed. The finite element simulation gives good agreement with the experimental results. Also the results of the current work indicate that increasing the temperature causes a local failure due to fusion of the node at which the temperature was applied.

Key Words: Space trusses; Test Setup; Fire Conditions; Finite Element Simulation.

1. INTRODUCTION

Space trusses are three dimensional structures assembled of linear elements. These elements are arranged in a way that ensures a three dimensional force transfer from the load application point to supports. Space structures have the ability of covering large areas. They attracted architects attention because they gave freedom in design and a pleasing appearance (Iffland 1982 [1], Iffland 1987 [2], Lan 1985 [3] and Lan 1999 [4]). Space truss connection has an important effect on the behavior and the cost of these structures. The MERO space truss joint system is a multidirectional system. This type of connection allows up to eighteen tubular members to be connected together through a spherical node at various angels. Another system called triodetic in which, bars are folded and cut to precise angles at their ends and inserted in special slots made in the nodes using pressure. One of the most common connections used for steel space trusses is the connections obtained by joining the end-flattened tubes with a single bolt. The advantages of such connections are reduced costs and fast assemblage of the truss, and their disadvantages are the eccentricities and stiffness reduction of the truss members (Schmidt et al. 1982 [5], Hanaor et al. 1989 [6], EL-Sheikh 1996 [7], De Andrade et al. 2005 [8] and El-Shami et al. 2016 [9]).

Space structures might face a brittle type of failure. This failure may be initiated by the overloading and consequent buckling of compression members (Mezzina et al. 1975 [10], Schmidt et al. 1976 [11], Collins 1981 [12], Smith 1988 [13] and El-Bakry 1995 [14]). Several studies have been done to improve the nonlinear behavior of space truss. Schmidt and Hanaor 1979 [15] introduced an artificial ductility to the compression member by building into it a Force Limiting Device (FLD). FLD is a device used to provide the strut (compression member) with an elastic-plastic load-deflection characteristic. El-Sheikh (1991) [16] increased the ductility of the space truss using over-strengthened top chord members. El-Sheikh (1991) [16], El- Sheikh and El-Bakry (1996) [17] and Shaaban (1997) [18] performed series of experimental tests and numerical analyses to improve the overall behavior of the space trusses using the composite action between the upper concrete slab and the top members of the truss. Eltaly (2010) [19] investigated the behavior of space trusses with top ferrocement slabs. Her results showed that the ferrocement slab decreases the force in the top chord members. Nawer, 2014 [20] improved the nonlinear behavior of space trusses using top sandwich panel. Top sandwich panel slab is considered one of the common types of slabs which are used to cover the steel structure for thermal insulation and light weight.

Amr M. Ibrahim, Ahmed A. Elshafey, Boshra A. Eltaly and Kamel S. Kandil "Structural Per..."

The incremental reduction in the strength of steel with increasing temperature is a well-known phenomenon which, in the event of fire, can lead to catastrophic collapse of structures. Bujakas 1998 [21] reported on the shape control of space structures, and stated that manufacturing, assemblage, and deployment errors, as well as thermal deflections, are recognized as major causes of shape distortion in large space designs. Chilton 2000 [22] showed that with the thermal movements associated with the normal changes in ambient temperature, it is desirable to provide appropriate sliding bearings to absorb expansions; otherwise, the expansion may induce high compressive forces in the space grid due to the rigid restraint. Yarza et al. 1993 [23] processed a mathematical model to find the consequent reduction in strength of the steel bars over a given time. The researcher stated that corner-supported space grids are most vulnerable in fire, due to the potential catastrophic failure of the diagonal web members immediately adjacent to the supports or to bending failure due to the collapse of the chord members. Alinia and Kashzadeh 2006a [24] studied the influence of the flexibility of substructures on the behavior of space truss domes, by predicting the thermal forces and stresses transmitted to the columns. They introduced a uniform thermal loading to six studied double layer space truss models. For each model, fifteen different support conditions plus the two extreme pinned and roller supports were studied. Alinia and Kashzadeh 2006b [25] then completed their studies by studying the effect of support flexibility on the behavior of the double layer space truss domes structures subjected to gradient and partial loadings.

In common with all steel structures, the effect of fire on space grids must be considered. The current work presented construction and testing of two space trusses to study the effect of fire on their structural performance up to failure. The space trusses tested were simulated by a finite element program, ANSYS software. The comparison between the results of the experimental and finite element models of these types of space trusses is presented.

2. EXPERIMENTAL WORK

2.1. Specimens Details

In the current work, two different corner supported double layer space trusses were constructed and tested under static loads and under fires up to failure. The two trusses have the same view as shown in Figure 1 and they are different in the overall dimensions (L, B and h), the members dimension and the type of connections. The first space truss (truss#1) has 3000 mm length (L), 3000 mm width (B) and 500 mm depth (h). All the members of truss#1 were tubular cross-section with 42.4 mm outer diameter and with 3.25 mm thickness. The members were connected together with MERO connections. The second tested truss (truss#2) has overall dimensions of 2850 x 2850 x 500 mm. All of the truss members of truss#2 were made of tubular cross-sections with 25 mm outer diameter and with 1.25 mm wall thickness. The four corner diagonal members of truss#2 were chosen to be a tubular cross-section with 32 mm outer diameter and with 1.5 mm wall thickness. All the members of truss#2 were connected together using the end-flattened connection.

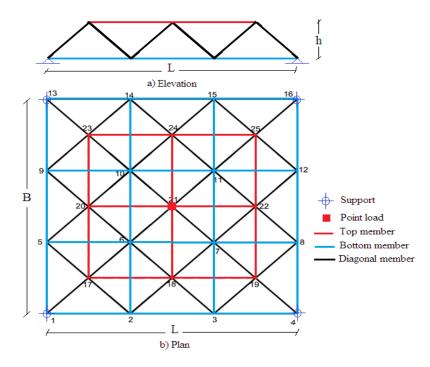


Figure 1: Tested trusses layout

2.2. Assembling Process of Space Trusses

The supporting frame of the tested trusses consists of four columns with SIB200 and with 800 mm length. These columns are supported on four reinforced concrete (RC) footings with 400 x 400 x 400 mm overall dimensions and each footing was reinforced by $5\Phi 8$ mm in the two directions as shown in Figure 2. The four footings were connected together by SIB120 steel beams in all directions. Also the columns were connected together in all directions by SIB120 beams as shown in Figure 3 to prevent the side sway. Assembling process of the space truss#1 members was carried out through four steps. The first step was the classification process of the truss members to upper, lower and diagonal members and the MERO connections to connections with 3, 4, 5, 6 and 8 holes. The second step was determination of the location of the lower MERO connections by drawing a layout for these connections on the supporting frame then the lower connections were inserted in their locations. At the third step, the lower chord members were connected to each other through these connections. The fourth step was determining the location of the MERO connections of the upper chord members then the diagonal members were connected to these connections after that the top members were joined to these connections as shown in Figure 4. Finally, all the connections were well tightened to increase the truss stiffness and to make the truss acts as one unit. The truss was supported on the supporting frame through four connections. Each connection consisted of pipe with 200 mm height. Two plates with 200 x 200 mm dimensions and 10 mm thickness were welded at the two ends of the pipe. Then the top plate was welded to the corner truss connection and the bottom plate was connected with the supporting frame column through four bolts as shown in Figure 5.

Truss#2 connections were made by flattening the ends of the member and assembling them with one bolt (M17 bolt) as shown in Figure 6. The lower truss members were assembled first followed by the upper chord members. Then the diagonal members were connected to the upper and lower chord members. Finally the truss placed on the supporting frame see Figure 7.

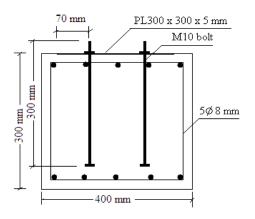


Figure 2: Footing details



Figure 4: Assembling process of truss#1 members



Figure 6: Truss#2 connection



Figure 3: Supporting frame



Figure 5: Connection between truss#1 and supporting frame



Figure 7: Assembling process of the truss#2 members

2.3. Testing of Individual Truss Members

Three specimens of each member of the two trusses were tested under compression force to obtain the buckling and post-buckling behavior of the truss members. The members were prepared by welding two stiff circular plates with 100 mm diameter and 30 mm thickness at the two ends. Each member was inserted in the supporting frame as shown in Figure 8. Tensile tests were conducted to determine the stress–strain curve of the truss members. Two specimens were taken from tubes with 1000 and 866 mm length. The specimens were prepared by adding steel bar with diameter 220 mm lengths at the two ends of the members. These steel bars were added to avoid flattening of the members ends under the gripping force. The test was carried out on the universal testing machine. The samples were placed in the machine and gripped by the machine ends as shown in Figure 9. Then the test was started by pulling the two ends of the member till its failure.



a) Compression







Buckling c) Yielding Figure 9: Member failure

For truss#1, the upper and lower chord member with 1000 mm length started to buckle at 110 kN axial load, while the diagonal member with 800 mm length started to buckle when the load reached to 250 kN and the member failure is presented in Figure 9. The relation between the load and the axial shortening is represented in Figure 10. The relation between the load and the axial shortening for truss#2 members are represented in Figure 11 and Figure 12. From the two figures, it can be shown that the chord member buckled at 27.08 kN axial load, the diagonal member buckled when the load reached to 23.05 kN load on the other hand the corner member buckled at axial load of 199.94 kN. The relation between stress and strain curve obtained from the tension test is indicated in Figure 13. From this curve, the modulus of elasticity, yield stress and ultimate strength of the truss were considered as 215 GPa, 380 MPa and 525 MPa, respectively.

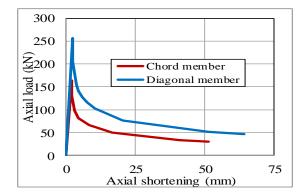


Figure 8: Truss member Tests

Figure 10: Load-axial shortening curve for chord and diagonal member of truss#1

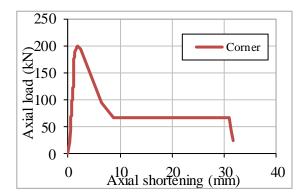


Figure 11: Load-axial shortening curve for corner member of truss#2

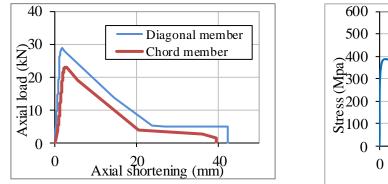


Figure 12: Load-axial shortening curve for chord and diagonal member of truss#2

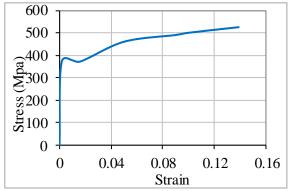


Figure 13: Stress-strain curve

2.4. Truss Testing Process

A hydraulic jack with 500 kN capacity and the displacement dial gauges with an accuracy of 0.01 mm were installed on the space truss as shown in Figure. 14. Also the strain was measured at selected members (see Figure. 15) by means of an extensometer. The testing process of truss#1 was performed in three stages. In the first stage, a concentrated load was applied at top mid joint of the truss (joint#21) with 5 kN load increment. In the second stage, 80 kN a constant static load was applied at top mid joint and an increase in temperature (from 100 °C to 400 °C) was applied at two lower joints of the truss joints (7 and 11). In the third stage, 80 kN a constant static load was applied at the top mid joint and a uniform increasing in temperature was applied at four lower joints of the truss joints (6, 7, 10 and 11).

The testing process of truss#2 was performed in two stages. A concentrated load was applied at top mid joint of the truss#2 (joint#21) with 5 kN load increment in the first stage. In the second phase, the four lower joints (6, 7, 10 and 11) were subjected to fire load and the truss was loaded by 20 kN constant static load applied at joint#21. In this test, the temperature increased from 300 °C to 900 °C with a 200 °C temperature increment. The time increment of temperature is 30 minutes.



a) Without fire



b) With fire

Figure 14: Test set up

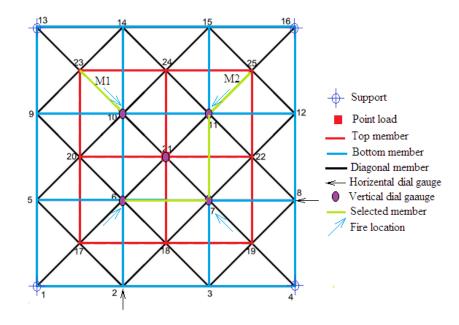


Figure 15: The location of displacement gauges, strain gauge and fire applied

3. FINITE ELEMENT SIMULATION

The two trusses tested were simulated using the finite element program (ANSYS Version 12). The members of truss#1 were represented in the FEM program as Link8 elements. Each element has three degree of freedom at each node; translations in x, y and z directions. This element is defined by two nodes, the cross-sectional area and its material properties. The truss was analyzed under the same conditions taken in the experimental test. The material properties were represented as the experiment results and the truss supports were considered as hinged supports. The truss was analyzed in three stages. In first stage, the truss was analyzed through series of linear static analyses with different static concentrated load at joint#21; 5 kN, 10 kN up to 80 kN applied load. In each series of linear analysis, the displacements and the forces in the members were recorded and used in the comparisons with the experimental results. The second stage was held by applying a fire load (increase in temperature) to the lower joint#7 and #11 and applying a load of 80 kN concentrated at joint#21. In the third stage, the truss was subjected to a uniform increase in temperature at lower joints (6, 7, 10 and 11) in addition to the static load with value 80 kN applied to the joint#21.

For truss#2, each member was simulated by two Link8 elements and COMBIN39 element as shown in Figure 16 (Eltaly 2010 [19]). Link8 element has three degree of freedom at each node; translations in x, y and z directions. COMBIN39 element has longitudinal or torsional capability in several dimensional applications. The element with longitudinal capability is a tension-compression element with three DOFs at each node (translations in the nodal x, y, and z directions). The element with torsional capability is a purely rotational element with three DOFs at each node; rotation about the nodal x, y, and z axes. Link8 element was adjusted to be very stiff such that it has no effect on the behavior of the truss model. The nonlinear material behavior was developed by COMBIN39 element with longitudinal capability. In order to make the model stable, anther Combine39 element with torsional capability is built on the COMBIN39 element with longitudinal. The additional COMBIN39 element should be defined with large stiffness to ensure the stability of the space truss element. COMBIN39 element was defined by two nodes and a nonlinear generalized force-deflection curve. This curve represents the nonlinear behavior of the truss member in the tension and compression. The force-deflection curve should be inserted such that deflections are increased from the compression to the tension. The last input deflection must be positive. Segments tending towards vertical should be avoided. The displacement control technique was used to obtain the behavior of the member up to failure. This technique was used because the strength of space trusses regularly degrades after reaching the plastic stage of the behavior. The truss supports were considered as hinged supports. In the case of the fire, the truss was subjected to a uniform increase in temperature at selected lower joints. The finite element simulation was presented in Figure 17.

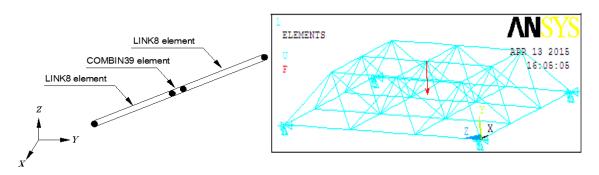


Figure 16: Modeling space truss members

Figure 17: Finite element simulation

4. RESULTS AND DISCUSSION

4.1.Truss#1

The results obtained from the finite element simulation for truss#1 were represented as relations between loaddisplacement, load-axial force, temperature-displacement and temperature-axial force for the selected joints and members (see Figure 15) were compared with the experimental results in the current section. For the first test, the relationship between the applied load and the corresponding mid-span deflection (at point#21) is shown in Figure 18. From this figure, it can be shown that the numerical load-central vertical deflection relationship is linear up to load of 90 kN with central deflection 3.145 mm. At this load, local failure at a corner joint (at the welding between the tube member and the cone of the member's end) occurred as shown in Figure 19. The collapse of this connection leads to failure of all the surrounding members and it leads to large deflection to all surrounding joints. Also from these figures, it can be observed that the failure occurred in the weak point of the member. The relation between load and axial force is presented in Figure 20. This relation is linear till the failure occurred at the location of the applied load.

Figures 21 to 27 present the results of the tested truss#1 for the second stage of the test as obtained from the experimental program and numerical model. From Figure 21, it can be clearly seen that the displacement was recorded 23.2 mm at temperature 200°C for point#7 from the FE simulation while the displacement for this point at the same temperature was measured as 20.88 mm from the experimental work. At temperature 300°C, the displacement at point#7 is 33.55 mm and 30.19 mm from FE simulation and experimental work; respectively and at temperature 400°C, the displacement for point#7 is 43.9 mm and 39.5 mm from FE simulation and experimental work; respectively. From Figure 22 and Figure 23, it can be concluded that the difference between the axial forces in the selected members as obtained from the experimental and FE simulation for the second test stage does not exceed 10%. Figures 24 and 25 showed local failures occurred in the truss joints at the joints subjected to the increase in temperature (at point#7 as a result of their location from the fire source. The deflection in joint#6 decreases by 77% than that measured for joint#7. Also the axial force in the two selected members (M#1&M#2) was calculated as presented in Figure 27. From Figure 27, it can be observed that the members near the fire source (M2) are subjected to a high stress than that one far away from the heat source (M1). Also from the results indicated in the two figures 26 and 27), it can be concluded that the truss members when half the truss was subjected to fire.

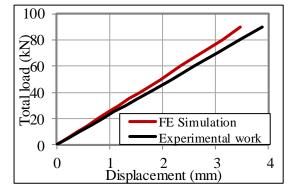


Figure 18: Load-deflection curves for the first test of truss#1 at joint#21



Figure 19: Collapse in the corner member from the first test of truss#1

Amr M. Ibrahim, Ahmed A. Elshafey, Boshra A. Eltaly and Kamel S. Kandil "Structural Per..."

For the third test stage, the failure in this case occurred at 400 °C temperature as a result of local failure in the truss joints as shown in Figures 28 and 29. The experimental test indicates that the deflection at all recorded joints (6, 7, 10&11) are the same and the axial forces in the two members (M1&M2) are equal. The relation between the displacement and temperature for the third test as obtained from the experimental and numerical results is illustrated in Figure 30. From this figure, it can be observed that the experimental and numerical curves are very close and the difference between their results do not exceed 15%. Figure 31 illustrated that by increasing the temperature to 200°C, the axial force becomes 99.69 kN and 110.77 kN from the experimental and numerical results; respectively and by increasing the temperature to 300°C, the axial force becomes 153.081 kN and 170.09 kN from the experimental and numerical results; respectively. Additionally when the temperature was increased to 400°C the axial force in the selected member#1 is 229.4 kN and 206.46 kN as obtained from the numerical and experimental results; respectively.

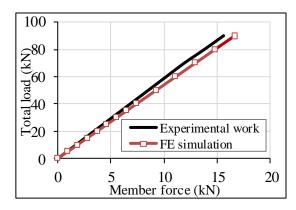


Figure 20: Axial force for M#2 form the first test of truss#1

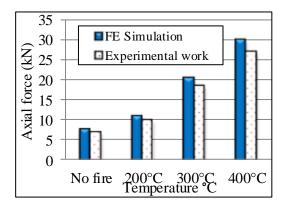


Figure 22: Axial force for M#1 form the second test of truss#1



Figure 24: Collapse in connections (7&11) from the

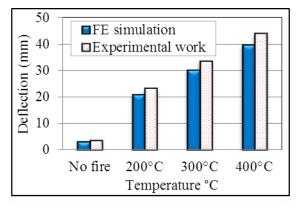


Figure 21: Temperature- deflection curve at joint#7 for the second test of truss#1

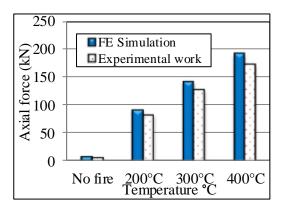


Figure 23: Axial force for M#2 form the second of test of truss#1

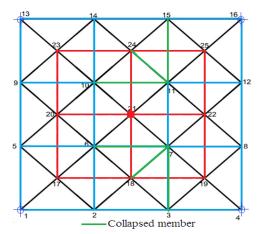


Figure 25: Collapsed members from the second test

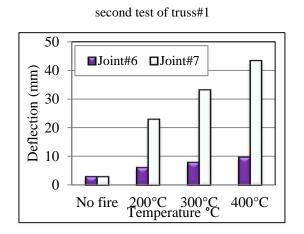


Figure 26: Experimental temperature-deflection curve for joint#6&7 from the second test of truss#1



Figure 28: Collapse in connections (20&23) from the third test of truss#1

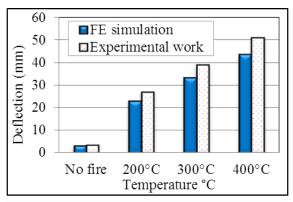


Figure 31: Temperature- deflection curves at joints#6 &7 form the third test of truss#1

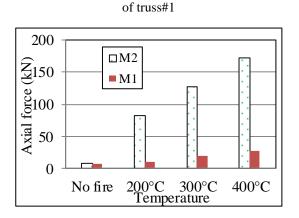


Figure 27: Experimental temperature-axial force curve for M#1&M#2 from the second test of truss#1

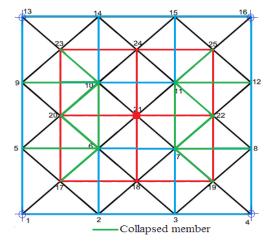


Figure 29: Numerical collapsed members from the third test of truss#1

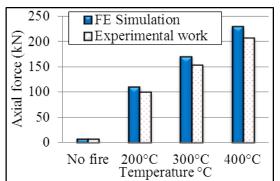


Figure 32: Axial force for M#1&M#2 form the third test of truss#1

4.2. Truss#2

In this section, the experiment results of the tested truss#2 without fire are presented and discussed. Figure 33 presents the load-deflection curves at joint#10. This figure indicated that the relation between load and corresponding deflection are linear till 70 kN total load and 22.43 mm as obtain from the experimental work. At this load, the failure occurred in the truss due to buckling of a top member that attached directly to the joint at which the load was applied (see Figure 34). After that, the truss was not able to carry more applied load so that the applied load decreases to be 48 kN. Also from Figure 33, it can be seen a good agreement between the results obtained from the experimental and numerical analysis especially in the linear zone. The ANSYS results showed that space truss#2 collapsed at 68.20 kN

Amr M. Ibrahim, Ahmed A. Elshafey, Boshra A. Eltaly and Kamel S. Kandil "Structural Per..."

total load and 25 mm corresponding deflection at joint#10 due to buckling of shown members in Figure 35. Then the applied load is decreased with large increasing in the deflection till 50.08 kN applied load and 40.04 mm corresponding deflection. Figure 36 shows the axial forces in members M3 and M4 (see Figure 15) as obtained from the experimental and the numerical analysis. The axial force at the end of the linear zone is to be 21.2 kN from the numerical analysis and is 21.82 kN from the experimental test. From the previous results, it is clear that the difference between two results does not exceed 5%.

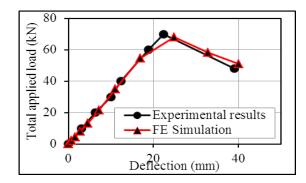


Figure 33: Load-deflection curves from the first test of truss#2

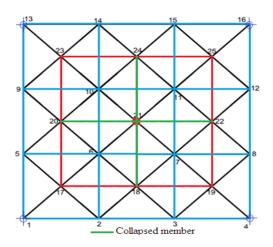
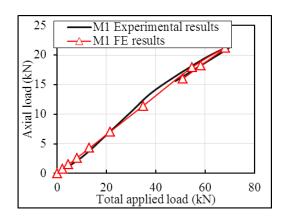


Figure 35: Numerical collapsed members from the first Figure 36: Axial force for M#3 form the first test of test of truss#2



Figure 34: Experimental collapsed members from the first test of truss#2



truss#2

The results of the studied space truss#2 under the effect of fire are presented in Figures 37 to 39. Figure 37 shows a comparison between the increase in temperatures (from 300°C to 900°C) and the deflection at joint#37 as obtained from the experimental work and numerical model. Figure 38 shows a comparison between the experimental work and numerical model results in term of axial forces in M#3 member with different temperatures. From the two figures, it can be concluded that the difference between the experimental and the numerical results do not exceed 10%. From Figure 37, one can observe that the truss deflections increase with increasing the applied temperature. On the other hand, Figure 38 shows that the axial forces in bottom member (M#3) increase by applying 300 °C temperature then the forces decrease by increasing the applied temperature. That is because the collapse in the truss started in appearing at applying 300 °C temperature as a local failure at the four joints at the source of the temperature as illustrated in Figure 39. This failure causes decreasing in the truss stiffness with the temperature increase. From these results, it can be concluded that the heat transfer from the joints to the members was very slow that causes buckling or yielding in the truss members.

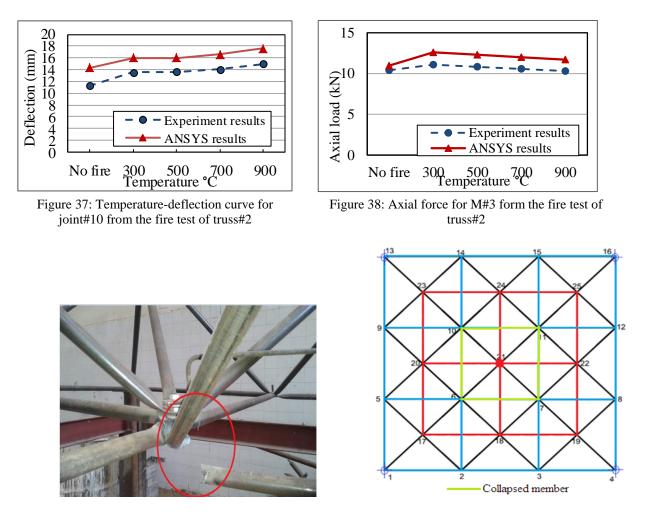


Figure 39: Collapsed members form the second test of truss#2

5. CONCLUSION

The current research presented three tests on the space truss to estimate the structural behavior of these structures under the effect of fire. In the first test, a concentrated load was applied at middle of the truss. The second test took place by applying a constant static load in addition to increase in temperature to two lower joints of the truss. The third stage test is similar to the second test except that a uniform increase in temperature was also applied to the four lower joints of the truss. In general good agreement can be clearly seen between the results obtained from the finite element program and the experimental test. Direct fire results in local failure due to the fusion of material and leads to sudden failure. From the results of the current experimental work and analytical model, it can be noted that:-

For the space truss#1

- From the experimental results of the space truss without fire, it is found that the truss collapsed locally in corner joints due to the high stiffness of the tested truss at 90 kN total applied load.
- From the two experimental tests of the fire, it is observed that the failure occurred is due to the loss of tension members that are directly attached to the heated joints, due to the increase in tension force (these members are separated from connection at the weld location due to the fusion of this zone).
- In the second model test in which half the truss were heated, the deflection of the heated joints increased than that joints which are far away from the heat source, this increase in deflection reached about 77%. It shows a significant increase due to the fire load.
- From the results of the second test, it can be seen that the axial force induced in the members attached directly to the heat source was so big in value (172.7 kN) than that induced in the members that are far away from the heat source which found to be nearly 30 kN (this shows the great effect of the fire on the truss).
- From the results obtained from the third test model, it is found that the deflection of the heated joints increased by a percentage that is nearly equal to 20% than that deflection obtained from the second model test at the same heated joints and by a percentage that equals nearly 75% than that deflection obtained from the first model test.

• Also from the results of the third test, it can be concluded that subjecting the four lower joints of the truss to the increasing temperature led to increasing of the axial force induced in the members that are attached directly to the heated joints from (172.7 kN) that obtained from the second test to (206.46 kN) at 400 oC temperature.

For the space truss#2

- The failure occurred in the truss#2 due to buckling of a top member that is attached directly to the joint at which the load was applied.
- After the occurrence of the buckling in member, brittle failure occurs in the truss.
- In the case of applying fire, the failure is a local failure at the four joints at the source of the temperature.

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